This Guideline provides the methods by which the thermal output of Intermediate and Large Renewable Thermal Generation Units (RTGUs) qualified for the Alternative Portfolio Standard (APS) shall be metered and how the meter readings are to be used in conjunction with the APS renewable thermal alternative energy credit formula to determine the number of Alternative Energy Credits (AECs) generated. This document is Part 2 of the Guideline on Metering and Calculating the Useful Thermal Output of Eligible Renewable Thermal Generation Units, Part 1 of the guideline can be found at the link below.¹ A Table of Contents, Table of Figures, and List of Tables can be found immediately following this section.

The purpose of this Guideline is to ensure uniform, accurate, reliable, and verifiable measurements of RTGU performance for determination of APS benefits, as appropriate to RTGU size and expense.

This Guideline is effective immediately upon issuance. However, the Department of Energy Resources (Department) may consider exceptions from the Guideline in the case of RTGUs that went into commercial operation prior to the issuance date, but not earlier than January 1, 2015.

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1. **Provisions in the Statute and Regulations**

The APS statute at M.G.L. Chapter 25A, Section 11F½(a)², as amended by Chapter 251 of the Acts of 2014, mandates the following as an eligible Alternative Energy Generating Source (emphasis added as italics):

(iv) any facility that generates useful thermal energy using sunlight, biomass, biogas, liquid biofuel or naturally occurring temperature differences in ground, air or water, whereby 1 megawatt-hour of alternative energy credit shall be earned for every 3,412,000 British thermal units of net useful thermal energy produced and verified through an on-site utility grade meter or other means satisfactory to the department;

Pursuant to the verification provision in that language, the APS regulations state the following at 225 CMR 16.05(4):

(a) **Metering Requirements.** The net Useful Thermal Energy output from an APS Renewable Thermal Generation Unit shall be metered according to the specifications laid out in the Department’s *Guideline on Metering and Calculating the Useful Thermal Output of Eligible Renewable Thermal Generation Units* and verified by an independent Third Party Meter Reader as defined in Rule 2.5(j) of the NEPOOL GIS Operating Rules and approved by the Department. The APS Alternative Generation Attributes reported to the NEPOOL GIS by an independent Third Party Meter Reader shall be the amount as specified in 225 CMR 16.05(1)(a)6.b. This amount will be inclusive of any netting of energy use by the APS Renewable Thermal Generation Unit as prescribed in 225 CMR 16.05(1)(a)6.b.iii., and the application of any multiplier in 225 CMR 16.05(1)(a)6.b.ii.

(i) An APS Renewable Thermal Generation Unit that uses more than one eligible technology in 225 CMR 16.05(1)(a)6.a. is required to use the same independent Third Party Meter Reader for all technologies.

(ii) Each APS Renewable Thermal Generation Unit is required to have its own individual NEPOOL GIS identification. An APS Renewable Thermal Generation Unit that uses more than one eligible technology in 225 CMR 16.05(1)(a)6.a. is required to have a NEPOOL GIS identification for each technology.

This Guideline specifies the manner by which the output of RTGUs can be verified through on-site meters which meet the minimum APS metering requirements as described in Section 3 of this document, or other means as specifically approved by the Department on a case by case basis.

2. **Applicability**

This document provides general guidance on the type, number, and location of meters specific to each type of RTGU eligible under the APS. The Department strongly suggests that information showing the number, type, and location of meters to be installed, be submitted to the Department for preliminary

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² The APS statute is available at [https://malegislature.gov/Laws/GeneralLaws/PartI/TitleII/Chapter25A/Section11F1~2](https://malegislature.gov/Laws/GeneralLaws/PartI/TitleII/Chapter25A/Section11F1~2). These were amended by sections 1, 2, 3, and 9 of Chapter 251 of the Acts of 2014.
review prior to the issuance of for bid or for construction designs and/or before procurement of the meters.

All direct measurements of energy are to be done by meters which comply with the requirements as set forth in Section 3 of this Guideline. The Department has determined that it is appropriate, practical, and non-burdensome to require that Large RTGUs be fully and directly metered and that a reduced level of direct metering combined with indirect metering is required for Intermediate sized RTGUs.

The ongoing operation of all RTGUs will be verified through means appropriate for each. This Guideline includes definitions of Small, Intermediate, and Large for the different eligible types of RTGUs as provided by the regulations and performance specifications for metering Intermediate and Large RTGUs. Additionally, the Department has determined that a subset of Large RTGUs that use biomass, biofuels, or biogas can be accurately metered by using fuel input recording, in combination with the RTGU’s energy conversion efficiency and fuel energy content.

Table 1 summarizes how RTGUs will be classified based on their capacity and defines the cut-off points for distinguishing between Small, Intermediate, and Large RTGUs. The size thresholds apply to the total combined capacity of the RTGU serving the thermal load. If an RTGU consists of several individual separate units, their capacities will be summed and the total capacity will be considered against the size threshold. In the case of a combination of solar thermal and other RTGUs, the thresholds will be applied separately to the solar and non-solar RTGUs.
Table 1: APS Renewable Thermal Generation Unit Classification

<table>
<thead>
<tr>
<th>Classification</th>
<th>Small</th>
<th>Intermediate(^1)</th>
<th>Large(^2)</th>
</tr>
</thead>
<tbody>
<tr>
<td>AEC Calculation Basis</td>
<td>Calculated net renewable thermal output</td>
<td>Calculated net renewable thermal output based on indirect metering</td>
<td>Calculated net renewable thermal output based on direct metering of fuel input</td>
</tr>
<tr>
<td>Solar Flat Plate / Evacuated Tube</td>
<td>Collector Surface &lt; 660 sq ft</td>
<td>Collector Surface Between 660 &amp; 4000 sq ft(^1)</td>
<td>-</td>
</tr>
<tr>
<td>Solar Hot Air / Sludge Dryer</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>Biomass Pellets/Chips</td>
<td>Output Based on Input Capacity &lt; 0.34 MMBtu/h</td>
<td>-</td>
<td>Capacity Between 0.34 &amp; 1 MMBtu/h</td>
</tr>
<tr>
<td>Biogas/Biofuels(^3)</td>
<td>-</td>
<td>-</td>
<td>Capacity &lt; 1 MMBtu/h</td>
</tr>
<tr>
<td>Electric Motor Driven Air Source Heat Pump</td>
<td>Output Capacity &lt; 0.134 MMBtu/h(^4)</td>
<td>-</td>
<td>Output Capacity Between 0.134 &amp; 1.0 MMBtu/h(^4)</td>
</tr>
<tr>
<td>Ground Source Heat Pump</td>
<td>Output Capacity &lt; 0.134 MMBtu/h(^5)</td>
<td>-</td>
<td>Output Capacity Between 0.134 &amp; 1.0 MMBtu/h(^5)</td>
</tr>
<tr>
<td>Engine Driven Air Source Heat Pump</td>
<td>Output Capacity &lt; 0.134 MMBtu/h(^4)</td>
<td>-</td>
<td>Output Capacity Between 0.134 &amp; 1.0 MMBtu/h(^4)</td>
</tr>
<tr>
<td>Deep Geothermal</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
</tbody>
</table>

1 Submetering of non-renewable fuel (e.g. gas or grid electricity) consumption will be used in conjunction with original equipment manufacturer performance data to calculate net thermal output for Intermediate size Air Source Heat Pumps (ASHP), Ground Source Heat Pumps (GSHP), and Solar Thermal RTGUs.

2 The thermal output and non-renewable fuel consumption are metered directly.

3 All Biogas/Biofuel RTGUs must apply via an aggregator as described in section 4(H) below.

4 Output heating capacity at entering source air temperature of 5°F

5 If Air-Conditioning and Heating Institute (AHRI) Certificate exists, output heating capacity as indicated on the AHRI Certificate at Full Load. If AHRI Certificate does not exist, use manufacturer’s rated output heating capacity as indicated below:

(a) For closed loop, water to water heat pumps: capacity at source entering water temperature of 32°F and load entering water temperature of 104°F If multiple ratings are shown under these conditions, use the source water and load water flow rate that results in the largest heating capacity.

(b) For open loop water to water heat pumps: capacity at source entering water temperature of 50°F and load entering water temperature of 104°F. If multiple ratings are shown...
under these conditions, use the source water and load water flow rate that results in the largest heating capacity.

(c) For closed loop, water to air heat pumps, capacity at source entering water temperature of 32°F and load entering air temperature of 70°F. If multiple ratings are shown under these conditions, use the source water and load air flow rate that results in the largest heating capacity.

(d) For open loop, water to air heat pumps, capacity at source entering water temperature of 50°F and load entering air temperature of 70°F. If multiple ratings are shown under these conditions, use the source water and load air flow rate that results in the largest heating capacity.

For the purpose of this Guideline, the definition of closed loop and open loop are as follows:

Close loop: Any water to air or water to water ground source heat pump system having no direct contact between the groundwater and the system fluid used for heat exchange.

Open loop: Any water to air or water to water ground source heat pump system which uses groundwater as the fluid for heat exchange.

3. **Metering Requirements and Formulae for Intermediate and Large RTGUs**

**A) General Principles**

(1) The British Thermal Unit (Btu) is a unit of thermal energy commonly used in the quantification of the capacity of a RTGU, as well as for the input and output energy of a system and/or component.

(2) All of the energy terms in the APS formulae for the determination of AECs are to be expressed in megawatt hours (MWh).

(3) 1 Btu = 1/3.412 watt hour; 1 MMBtu = 1,000,000 Btu = 1/3.412 MWh

(4) Net useful heat is the thermal energy that is transferred to a facility and/or process load and is equal to the thermal energy supplied to the load from the RTGU minus thermal energy returned from the load to the RTGU minus any parasitic thermal energy.

(5) All electricity supplied by the ISO-NE grid to a RTGU is considered to be non-renewable fuel and must be subtracted from the net useful heat generated. The amount of non-renewable source fuel per MWh of grid electricity consumed by a RTGU is equal to the MWh electricity consumed at the site divided by the most recently published ISO-NE marginal grid efficiency, which at the date of this version of this Guideline is 0.44 MWh source fuel/MWh electricity delivered.
(6) Parasitic Energy

Parasitic energy is defined as the electricity or thermal energy required to operate any auxiliary component or system of the RTGU, where auxiliary denotes a component and/or sub-system that does not directly generate useful thermal energy, but whose operation is required in order for the generation of Useful Thermal Energy to occur. Examples of auxiliary components are:

(a) Boiler feedwater pumps

(b) Combustion air supply fans

(c) Biomass boiler stokers

(d) Solar thermal collector fluid circulating pumps

Parasitic electricity is applicable only to RTUGs which also generate electricity (i.e. combined heat and power CHP systems)

Parasitic thermal energy may be applicable to all RTGUs; however it is typically limited to RTGUs which generate steam.

In general, components such as pumps, fans, blowers, etc. that may be installed and operated in conjunction with an RTGU whose function is to distribute the thermal energy generated by an RTGU to the useful thermal loads are not considered as auxiliary and the energy required to operate them is not metered or included in the determination of AECs. The exception is any RTGU which is located more than 500 ft. from the point of connection with a thermal load or with the thermal hosts’ distribution system being supplied by the RTGU.

(7) Quantification of Parasitic Electricity

Any metering of grid source electrical energy, required by this Guideline, shall be done using electric kilowatt hour (kWh) meters which conform to the requirements included in these guidelines. The minimum thresholds for direct metering are:

(a) Solar Thermal – No minimum

(b) Air, Ground, and Water Source Heat Pumps – Not Applicable

(c) Biomass, Liquid Fuel and Biogas – The parasitic electrical energy (kWh) for any auxiliary load that is rated at more than 0.75 kW (1 Horsepower) shall be metered using a revenue-grade kWh meter as specified in Section 3(F) of this Guideline. Auxiliary systems or components fed as a group by a single electrical service can be metered by a single kWh meter installed in that service (e.g. in the feed to a motor control center). Exceptions will considered by the Department on a case-by-case basis.
(8) **Quantification of Parasitic Thermal Energy**

All efforts should be made to locate a system’s Btu meters such that the consumption of parasitic thermal energy is netted out. In the event that this cannot be accomplished the parasitic thermal energy of any auxiliary component with a demand above a To Be Determined value during nominal operating conditions will require a separate Btu meter.

(9) **Non Useful Heat**

Thermal energy that is rejected to a heat sink (e.g. the air, ground, surface, or storm water) is not considered to be Useful Thermal Energy and must be accounted for in the location of Btu meter instruments as well as in the determination of the net metered useful energy.

Wherever possible, the components of Btu meters should be located such that they do not count heat rejected to a heat sink in the heat being metered. If this is not possible, separate Btu metering will be required to measure the heat rejected to a heat sink and this energy shall be subtracted from the total metered Btus.

(10) **General Guidance for Locating Btu Meters**

(a) Whenever possible, Btu meters should be located at a point before the interconnection with the load’s thermal distribution system (i.e. on the RTGU side and not on the load side).

(b) Whenever possible, Btu meters should be located before any point of connection with a non-useful heat load, such as a radiator of cooling tower that rejects excess heat, before delivery to the distribution system, or rejection of excess heating systems.

(c) When a RTGUs is located more than 500 ft from the point of connection with a thermal load, the Btu meter(s) must be located within 30 feet from the point of connection to the thermal load.

*Note: For this case pumping energy to circulate heat transfer fluid between the central RTGU and the point of connection with each remote building or self-contained load is to be considered as either grid electricity, or in the case of a Combined Heat and Power (CHP) RTGU, as parasitic.*
Figure 1 Diagram of a generic central RGTU with Distributed Thermal Loads

(11) **Measuring The Net Renewable Thermal Energy Transferred to a Facility’s Heating or Cooling Thermal Load**

(a) Air or Heat Transfer Fluids (including aqueous mixtures): based on mass flow, temperature, and specific heat

(b) Steam: based on mass flow and specific enthalpy (temperature & pressure)

**B) General Formulae for the Quantification of Alternative Energy Certificates for RTGUs**

(1) **Systems Using Only a Single Eligible Renewable Fuel**

\[
\text{AECs} = (UH - NUH - P_e - P_{th} - G/0.44) \times M \\
\text{or} \\
\text{AECs} = (H_{\text{supply}} - H_{\text{return}} - NUH - P_e - P_{th} - G/0.44) \times M
\]

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

Where:

\(UH\) = Net useful heat

\(NUH\) = Non-useful heat
\[ H_{\text{supply}} = \text{Thermal energy supplied from the RTGU to a useful load} \]
\[ H_{\text{return}} = \text{Thermal energy returned from a useful load to the RTGU} \]
\[ P_e = \text{Parasitic electricity (applies only to combined heat and power (CHP) RTGUs)} \]
\[ P_{\text{th}} = \text{Parasitic thermal energy} \]
\[ G = \text{Grid supplied electrical energy} \]

\[ M = \text{The multiplier assigned to a specific RTGU technology per the Department’s Guideline on AEC Multipliers for Renewable Thermal Generation Units} \]

(2) **System Using an Eligible Renewable Fuel and a Non-Renewable Fuel Which are Not Blended (i.e. Co-fired)**

Per formula in 3(B)(1) above, AECs are based only on metered readings for the period during which the generating system is operating on an eligible renewable fuel.

(3) **System Using an Eligible Renewable Fuel and a Non-Renewable Fuel Which are Blended**

Per formula in 3(B)(1) above, with the following modification:

\[ \text{AECs} = \left( \frac{RF}{RF + NRF} \right) \times \left\{ (H_{\text{supply}} - H_{\text{return}} - NUH - P_e - P_{\text{th}} - G/0.44) \times M \right\} \]

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

Where:

\[ UH = \text{Net useful heat} \]
\[ NUH = \text{Non-useful heat} \]
\[ H_{\text{supply}} = \text{Thermal energy supplied from the RTGU to a useful load} \]
\[ H_{\text{return}} = \text{Thermal energy returned from a useful load to the RTGU} \]
\[ P_e = \text{Parasitic electricity (applies only to combined heat and power (CHP) RTGUs)} \]
\[ P_{\text{th}} = \text{Parasitic thermal energy} \]
\[ G = \text{Grid supplied electrical energy} \]
\[ RF = \text{Renewable fuel} \]
\[ NRF = \text{Non-renewable fuel} \]
Notes:
1) “Renewable Fuel” denotes an eligible renewable fuel.
2) RTGUs using non-renewable fuels will be required to complete a fuel plan, which will be an appendix in the Statement of Qualification Application (SQA) for RTGUs.

B) Minimum Efficiency Standard for Large Wood Fueled RTGUs

All Large wood fueled RTGUs must have an overall net efficiency of forty percent (40%) or higher. For the purpose of this standard, the definition of the efficiency shall be:

Efficiency = (Number of AECs) / (MWh of wood fuel consumed for each month of the as-submitted projected average year)

Where the AECs are determined using the APS general formulae as shown above.

Approval of a Large wood fueled RTGU’s Statement of Qualification Application (SQA) by the Department is contingent upon the review and approval of information provided in the SQA, which must demonstrate the RTGU meets this minimum efficiency standard.

Approval of AECs as submitted to the NEPOOL Generation Information System (NEPOOL GIS) for a given quarter shall be contingent upon the metered average efficiency, as defined in the definition of efficiency above. The MWh of wood fuel consumed for the given quarter is to be determined per the method and protocol described in Section 3 of the APS Guideline on Biomass, Biogas, and Biofuels for APS Renewable Thermal Generation Units.

C) Acquisition, Recording, Storing, and Transmittal of Metered Data

All RTGUs must include a Data Acquisition System (DAS) which must meet the following minimum functional criteria:

(1) Input: Collect less than or equal to five-minute interval input data from each of the APS metering instruments and/or devices.

(2) Storage: Store up to 100 days cumulative input data.

(3) Output: Have remote electronic access to time stamped data of each input in five minute intervals that can be provided as a comma separated values (.csv) file.

The DAS may be a stand-alone dedicated unit or can be integrated into an existing system.

D) Standards for APS RTGU Meters

All meters required by the APS must meet and conform to all applicable laws, ordinances, codes, regulations, and standards, must be of revenue grade accuracy, quality, and reliability, and must have the capability to generate and transmit a signal to the system DAS.
### E) Thermal (Btu) Energy Meters

**Table 2. Thermal Energy Meter Requirements for Steam**

<table>
<thead>
<tr>
<th>Line Size</th>
<th>Btu Meter System Components</th>
<th>System Field Accuracy</th>
<th>Re-Calibration Interval</th>
<th>Notes</th>
</tr>
</thead>
<tbody>
<tr>
<td>All</td>
<td><strong>Btu Computer:</strong> Automated real-time computation and totalizer</td>
<td>—</td>
<td>—</td>
<td>—</td>
</tr>
<tr>
<td>≥8”</td>
<td><strong>Flow Sensor:</strong> Orifice Plate with Differential Pressure Element and Transmitter</td>
<td>Only with superheated Steam</td>
<td>±3% Annual</td>
<td>1) Perform an annual inspection of the flow sensor orifice plates and check for wear and distortion beyond the OEMs specifications as a part of the annual recalibration procedure. 2) If a significant percentage of flow occurs at flow rates below the flow sensors minimum guaranteed full accuracy flow rate, both the flow and Btus may be undercounted. This can be addressed by installing a two meter manifold with the meters sized to cover the entire expected range of flow rates. Consult the flow meter provider for design and installation details.</td>
</tr>
<tr>
<td>&lt; 8”</td>
<td><strong>Flow Sensor:</strong> Vortex Shedding Tube</td>
<td>Only with superheated Steam</td>
<td>±3% Biennial</td>
<td>—</td>
</tr>
</tbody>
</table>
### Table 3. Thermal Energy Meter Requirements for Hot Water

<table>
<thead>
<tr>
<th>Btu Meter System Components</th>
<th>Btu Meter Field Accuracy</th>
<th>Re-Calibration Interval</th>
<th>Notes</th>
</tr>
</thead>
<tbody>
<tr>
<td><strong>Flow Sensor:</strong> In-line Ultrasonic Flow Tube (no strap-on) or Magmeter</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Thermal Sensors:</strong> Installed in thermowells</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Btu Computer:</strong> Automated real-time computation and totalizer</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>±3%</td>
<td>Biennial</td>
<td>No turbine or impellor based flow sensors</td>
<td></td>
</tr>
</tbody>
</table>

### Table 4. Thermal Energy Meter Requirements for Air

<table>
<thead>
<tr>
<th>Btu Meter System Components</th>
<th>Btu Meter Field Accuracy</th>
<th>Re-Calibration Interval</th>
<th>Notes</th>
</tr>
</thead>
<tbody>
<tr>
<td><strong>Flow Sensor:</strong> a) In-duct Differential Pressure Measuring Airflow Station b) Pitot Tube</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Thermal Sensors:</strong> Installed in the air flow stream</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Btu Computer:</strong> Automated real-time computation and totalizer</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>±3%</td>
<td>Annual</td>
<td>Clean and inspect orifices as a part of the annual calibration procedure</td>
<td></td>
</tr>
</tbody>
</table>

### Table 5. Thermal Energy Meter Requirements for Refrigerant

<table>
<thead>
<tr>
<th>Btu Meter System Components</th>
<th>Btu Meter Field Accuracy</th>
<th>Re-Calibration Interval</th>
<th>Notes</th>
</tr>
</thead>
<tbody>
<tr>
<td><strong>Liquid Flow Sensor:</strong> Full Flow Ultrasonic (transit time)</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Thermal Sensors:</strong> RTD or Thermocouple Installed in thermowells</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Btu Computer:</strong> Automated real-time computation and totalizer</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>±3%</td>
<td>TBD</td>
<td>Care must be taken to avoid flashing of the hot liquid to vapor, which will affect the operation of expansion valves</td>
<td></td>
</tr>
</tbody>
</table>

**F) Fuel Meters**

In the event that one or more non-renewable fuels will be blended with one or more renewable fuels, the direct metering of each fuel is required.
Table 6. Fuel Meter Requirements for Natural Gas

<table>
<thead>
<tr>
<th>Meter Type</th>
<th>Flow rate Range</th>
<th>Accuracy</th>
<th>Minimum Frequency of Calibration</th>
<th>Other</th>
</tr>
</thead>
<tbody>
<tr>
<td>All Meters</td>
<td>2% ≤ Flowrate ≤ 100% Max DF Where Max DF is Maximum System Design Flow Rate</td>
<td>± 2%</td>
<td>Annual</td>
<td>See Notes</td>
</tr>
</tbody>
</table>

Notes:
1) All diaphragm meters must conform to the current American National Standard Institute (ANSI) B109.2 standard
2) All rotary meters must conform to the current ANSI B109.3 standard
3) The default Higher Heating Value (HHV) for pipeline natural gas is 1,030 Btu per Standard Cubic Foot (SCF). Alternate HHVs may be proposed as documented by the fuel supplier
4) All volumetric measurements must be adjusted to SCF (i.e. they must be temperature and pressures corrected). Meter models that are auto pressure and temperature compensated will meet this requirement. Meters with a settable fixed value pressure compensation may be used if installed downstream of a pressure regulator. The applicant may propose an alternate method for converting the metered flow to SCF units
5) Thermal Diffusion Meters (TDM) are not approved
6) A pipeline gas meter furnished and installed as part of a dedicated gas line to the RTGU system by the site’s gas utility will be accepted as an approved natural gas fuel meter

Table 7. Fuel Meter Requirements for Biogas

<table>
<thead>
<tr>
<th>Btu Meter System Components</th>
<th>Btu Meter Field Accuracy</th>
<th>Recalibration &amp; Inspection Frequency</th>
<th>Notes</th>
</tr>
</thead>
<tbody>
<tr>
<td>Gas Flow Sensor: Full Flow Averaging Pitot</td>
<td></td>
<td></td>
<td>Both tuned laser or infrared based optical gas analyzer technologies are acceptable</td>
</tr>
<tr>
<td>Real time gas analyzer</td>
<td>±3%</td>
<td>Annual</td>
<td></td>
</tr>
<tr>
<td>Btu Computer: Automated real-time computation and totalizer.</td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

G) Electric (kWh) Meters

Electric (kWh) meters shall be revenue grade and shall:

1) Be certified as meeting American National Standard Institute (ANSI) Standard C12.20
2) Have a kW and kWh remote output signal with an output signal interval of not more than once per minute
3) Have either a non-resettable or password protected cumulative kWh register.
H) Accuracy of Thermal and Fuel Metering

Thermal energy and fuel must be metered per the tables in 3(E) and 3(F) above, with a possible future modification per the issuance of the American Society for Testing and Materials (ASTM) Heat Meter Technology Standard WK37952 that is currently under development under the leadership of the United States Environmental Protection Agency (EPA).

I) Metering Requirements and Renewable Thermal AECs Formulae for Intermediate and Large Solar Hot Water RTGUs

(1) Major System Components

Intermediate and Large Solar Hot Water RTGUs include, but are not limited to, the following major components:

(a) A solar thermal energy collector system

(b) One or more unfired water storage tanks that supply water, pre-heated by solar energy only, to a primary (fired) heating system

(c) Heat exchanger(s) which transfer energy from the solar collector heat transfer fluid circulating loop to the volume of water in the pre-heat storage tank (not shown in Figure 2 below)

(d) Electric motor driven pump(s) which circulate a heat transfer fluid in a closed loop between the solar collectors and the collector fluid-to-unfired storage tank heat exchanger

(e) Automatic pump controller(s) which start and stop the circulating pumps based on a preset temperature difference between the temperature of the unfired storage tank and the temperature of the solar collectors

(2) Meter Systems for Intermediate and Large Solar Hot Water RTGUs

All Intermediate and Large solar hot water RTGUs must have Btu and kWh meters as shown in Figure 2, Figure 3, Figure 4, and Figure 5.

(3) Data Acquisition System(s)

(a) A DAS that meet the requirements in Section 3(D) above that records, stores, and transmits time stamped Btu meter readings, and the run times of the collector heat transfer fluid circulating pump(s)

These are based on the generic configurations as shown in Figure 2, Figure 3, Figure 4, and Figure 5. The Department will evaluate metering plans and submittals based on alternative configurations on a case-by-case basis.
Figure 2. Simplified Schematic of a Typical Intermediate Size, Evacuated Tube, Solar Thermal RTGU

Figure 3. Location of Pressure Indicating Gauges Up and Downstream From the Collector Heat Transfer Circulating Pumps
(4) AEC Formula for Intermediate Evacuated Tube, Solar Thermal RTGUs

\[ \text{AECs} = (\text{UH} - \frac{G}{0.44}) \times M \]

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

The rate of net useful thermal transferred to the load at any time is:

\[ \text{HTFF} \times \text{HTF density (lbs/gal)} \times \text{Cp} \times (\text{HTST} - \text{HTFRT}) \]

Where:

\( \text{HTF} = \) The heat transfer fluid that circulates between the collector array and the loop to storage tank heat exchanger (not shown)

\( \text{HTFF} = \) HTF flowrate (gpm)

\( \text{Cp} \text{ HTF} = \) The constant pressure specific heat of the HTF (Btu/lb-°F)

\( \text{HTST} = \) HTF supply temperature (°F)

\( \text{HTRT} = \) HTF return temperature (°F)

\( G = \) Grid supplied electrical energy

\( M = \) the current APS multiplier for flat plate evacuated tube solar hot water RTGU in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

(5) Metering of Intermediate, Evacuated Tube Collector, Solar Thermal RTGUs

Refer to Figure 2, Figure 3, and Figure 4 in Section 3(I)(3) and Section 3(I)(5).

(a) The temperatures heat transfer supply temperature (HTST) & heat transfer return temperature (HTRT) will be measured directly with thermal sensors located at the collector array and the grid supplied electrical energy (G) must be directly metered.

(b) The heat transfer fluid (HTF) flow will be indirectly metered using the manufacturer’s performance data for the specific pump. After the RTGU is commissioned and the HTF flows are balanced and set, the following data will be obtained and recorded for each of the HTF circulating pumps:

(i) The inlet and discharge pressure (psig) (for location of the pressure gauges, see Figure 3)
(ii) The average operating kW over a five minute interval, after full flow is established. A copy of a report issued by the pump supplier on company letterhead showing the system flow rate in pounds of HTF per hour for each pump is to be provided with the statement of qualification application. A copy of the pump performance curve showing the system pump design point (see Figure 4 below) must be included in the report.

(c) Based on the pump supplier’s determination of the heat transfer flowrate (HTFF) for each pump, the flow for each pumped HTF loop will be:

\[
\text{HTF flow (lbs) = Sum for all loops}
\]

Where the flow for each pump = HTF flow (lbs/hr) * kWh/kW pump

Where:

\[
\text{kW pump = kW pump as recorded in (b) above.}
\]

OR

The installer may use an approved temporary flow meter to measure and record the system flow rate for each pumped HTF circulating loop.

**Figure 4. Example of a Typical Pump Curve with Showing the Design Operating Point**
(6) Metering of Large Solar Thermal RTGUs

Figure 5. Simplified Schematic of a Typical Large, Evacuated Tube, Solar Thermal RTGU

Note: System heat exchangers not shown for clarity

(7) AEC Formula for Large Evacuated Tube Collector Solar Thermal RTGUs

\[ \text{AECs} = (UH - G/0.44) \times M \]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh
The renewable heat transferred over any interval is:

\[
\text{(CMWF} \times 8.34 \text{ (lbs/gal)} \times (\text{TSHW} - \text{TCMW})) / 3,412,000 \text{ (Btu/MWh)}
\]

Where:

CMWF = City cold makeup water flow rate (gallons)

TCMW = Temperature of the city cold makeup water supply (°F)

TSHW = Temperature of the solar hot water supply to the fired hot water heating system (°F)

G = Grid supplied electrical energy

M = the current multiplier for flat plate evacuated tube solar hot water RTGUs in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

(8) **Direct Expansion (DX) Air Source Heat Pumps (ASHPs)**

Both Intermediate and Large DX ASHPs will typically be a self-contained unit with a warm forced air output. The major components of a typical DX ASHP include, but are not limited to the following:

(a) Controls

(b) A closed refrigerant sub-system including:

(i) Compressor(s)
(ii) Evaporator(s)
(iii) Condenser(s)
(iv) JT expansion valves
(v) Refrigerant side tubing, instruments and control devices

(c) An air side sub-system including:

(i) Fans
(ii) Louvers and dampers
(iii) Filters
(iv) Air flow related internal ducting, instruments, and control devices
(9) Metering for Intermediate, ASHPs

Figure 6. Metering Diagram for an Intermediate, DX ASHP

(10) AEC Formula for Intermediate, DX ASHPs

\[ AECs = ((COPw \times NRF) - NRF) \times M \]

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh.*
Where:

\[ \text{COPw} = \text{The climate weighted heating coefficient of performance (COP) for the typical meteorological year (TYM) heating season as provided by the original equipment manufacturer} \]

\[ \text{NRF} = \text{The non-renewable fuel (e.g. grid electricity, or other) consumed by the RTGU} \]

\[ M = \text{the current multiplier for ASHPs in the Department’s } \textit{Guideline on AEC Multipliers for Renewable Thermal Generation Units} \]

When the non-renewable fuel is grid electricity:

\[ \text{NRF} = \frac{G}{0.44 \times F} \]

Where:

\[ G = \text{Grid supplied electrical energy} \]

\[ F = \text{The factor applied to the 0.44 grid efficiency to account for the improvement in the ISO-NE grid efficiency during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act} \]

(11) \textit{Metering of Large, DX ASHPs}
Figure 7. Metering Diagram for a Large, DX ASHP

Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for DX ASHPs are shown

(12) AEC Formula for Large, DX ASHPs

\[ \text{AECs} = \text{RH} \times \text{M} \]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh

Where:

\( \text{RH} \) = Renewable heat transferred from the ambient air to the system and is directly metered

The rate of renewable heat generated at any time is:

\[ (\text{EAF}) \times (\text{Cp}) \times (\text{ETA-LTA}) / 3.412E6 \text{ (Btu/MWh)} \]
Where:

\[ \text{EAF} = \text{Outside air flow rate (lbs/hr) through the Evaporator(s)} \]

\[ \text{Cp} = \text{Specific heat of the outside air (Btu / lb-deg F)} \]

\[ \text{ETA} = \text{Temperature of the outside air entering the evaporator section (°F)} \]

\[ \text{LTA} = \text{Temperature of the outside air leaving the evaporator section (°F)} \]

(13) **Variable Refrigerant Flow (VRF) ASHPs**

An Intermediate or Large VRF ASHP will typically be a split system consisting of an outdoor unit which generates hot liquid refrigerant coupled with an indoor distribution system connected with one or more air handler or fan coil terminal units. A typical VRF ASHP includes, but is not limited to the following principal components:

(a) Controls

(b) A closed refrigerant sub-system including:

   (i) Compressor(s)
   (ii) Evaporator(s)
   (iii) Condenser(s)
   (iv) Refrigerant expansion valves
   (v) Tubing, instruments and control devices

(c) An air side sub-system including:

   (i) Air handler unit(s)
   (ii) Fan coil terminal units
(14) Metering for Intermediate, VRF AHSPs

Figure 8. Metering Diagram for an Intermediate, VRF ASHP

Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for VRF ASHPs are shown.

(15) AEC Formula for Intermediate ASHPs

\[ \text{AECs} = ((\text{COPw} \times \text{NRF}) - \text{NRF}) \times M \]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh

Where:

COPw = The climate weighted heating coefficient of performance (COP) for the typical meteorological year (TYM) heating season as provided by the original equipment manufacturer

NRF = The directly metered, non-renewable fuel (e.g. grid electricity, or other) consumed by the RTGU.

When the non-renewable fuel is grid electricity:

\[ \text{NRF} = \frac{\text{G}}{0.44 \times \text{F}} \]

G = Grid supplied electrical energy
F = The factor applied to the 0.44 grid efficiency to account for the improvement in the ISO-NE grid efficiency during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act

M = the current multiplier for ASHPs in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

(16) Metering for Large, VRF AHSPs

Figure 9. Metering Diagram for a Large, VRF ASHP

Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for VRF ASHPs are shown.
(17) **AEC Formula for Large, VRF ASHPs**

\[ \text{AECs} = \text{RH} \times M \]

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

Where:

\( \text{RH} = \) Renewable heat transferred from the ambient air to the system and is directly metered

The rate of renewable heat generated at any time is:

\[
(\text{SRF}) \times (\text{Cp}) \times (\text{RST} - \text{RRT}) / 3.412E6 \text{ (Btu/MWh)}
\]

Where:

\( \text{SRF} = \) Supply refrigerant flow (lbs/hr)

\( \text{Cp} = \) Specific heat of the refrigerant supply (Btu/lb-deg F)

\( \text{RST} = \) Refrigerant supply temperature (°F)

\( \text{RRT} = \) Refrigerant return temperature (°F)

*Note: If the \( \text{Cp} \) of the RTGU refrigerant varies significantly as a function of the refrigerant temperature, the integration of this value over time will be done by the Btu computer software engine in time steps of less than or equal to five minutes each using a look up table or function as provided by the supplier of the refrigerant.*

(18) **Ground Source Heat Pumps (GSHP)**

A typical GSHP includes, but is not limited to the following major components:

(a) Controls

(b) A closed refrigerant sub-system including:

   (i) Compressor(s)
   (ii) Evaporator(s)
   (iii) Condenser(s)
   (iv) Refrigerant expansion valves
   (v) Tubing, instruments and control devices

(c) An air side sub-system including:

   (i) Fans
   (ii) Louvers and dampers
   (iii) Filters
(iv) Air flow related internal ducting, instruments, and control devices

(d) A well field

(e) A well field to heat pump circulating water loop including pump(s), piping, instruments and control devices

**Figure 10. Metering Diagram for an Intermediate GSHP**

Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for an Intermediate size GSHPs are shown.

(19) **AEC Formula for Intermediate GSHPs**

\[ \text{AECs} = ((\text{COPw} \times \text{NRF}) - \text{NRF}) \times M \]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh

Where:

\( \text{COPw} = \) The climate weighted heating coefficient of performance (COP) for the typical meteorological year (TYM) heating season as provided by the original equipment manufacturer
NRF = The directly metered non-renewable fuel (e.g. grid electricity, or other) consumed by the RTGU

*Note: Any NRF consumed by the GSHP to well field circulating pumps must be directly metered and included in the total NRF.*

When the non-renewable fuel is grid electricity:

NRF = \( G / (0.44 \times F) \)

\( M = \) The current multiplier for GSHPs in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

Where:

\( G = \) Grid supplied electrical energy

\( F = \) The factor applied to the current 0.44 ISO-NE marginal grid efficiency to account for the improvements during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act
(20) Metering of Large GSHPs

Figure 11. Metering Diagram for a Large GSHP

![Diagram of a Large GSHP](image)

Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for GSHP RTGU are shown.

(21) AEC Formula for Large GSHP

\[
\text{AECs} = (\text{RH} - \text{NRF}) \times M
\]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh

Where:

\text{RH} = \text{Renewable heat transferred from the ambient air to the system and is directly metered}
The renewable heat transferred over any interval is:

\[(SGWF \times Cp \times GWST - GWRT) / 3.412E6 \text{ (Btu/MWh)}\]

Where:

- \(GWSF\) = Ground water supply flow (lbs)
- \(Cp\) = Specific heat of the circulating well fluid (Btu/lb-deg F)
- \(GWST\) = Ground water supply temperature to the GSHP (°F)
- \(GWRT\) = Ground water return temperature from the GSHP (°F)
- \(NRF\) = The directly metered non-renewable fuel (e.g. grid electricity, or other) consumed by the RTGU

*Note: Any NRF consumed by the GSHP to well field circulating pumps must be directly metered and included in the total NRF.*

When the non-renewable fuel is grid electricity:

\[NRF = G / (0.44 \times F)\]

Where:

- \(G\) = Grid supplied electrical energy
- \(F\) = The factor applied to the current 0.44 ISO-NE marginal grid efficiency to account for the improvements during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act
- \(M\) = the current multiplier for GSHPs in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

(22) **Fired RTUGs**

“Fired” denotes that the useful heat is generated from the combustion of eligible fuels in the RTGU

*Notes: Fuel cells are included in this category. Any RTGU using a wood biomass fuel must comply with the added requirements as set forth in the APS Guideline on Biomass, Biogas, and Biofuels for APS Renewable Thermal Generation Units.*

Combustion modes for Fired RTUGs (all fuels must be eligible fuels) include the following:
(a) Combustion of a single renewable fuel

(b) Co-firing of more than one renewable fuel

(c) Co-firing of one or more renewable fuels and one or more non-renewable fuel

(d) Blending of one or more renewable fuels

(e) Blending of one or more renewable fuels with one or more non-renewable fuels

Note: Co-firing denotes switching fuels without blending.

(23) Fired RTGUs Which are Combined Heat And Power (CHP) Systems

An RTGU which co-generates electricity and useful heat is designated as a CHP RTGU and may qualify for Massachusetts Portfolio Standard Programs and earn credits in one of two different ways:

(a) A CHP RTGU may qualify as a Renewable Portfolio Standard (RPS) Class I generator and as an APS RTGU. In this case:

(i) The net MWh electricity generated by the CHP RTGU earns Class I Renewable Energy Credits (RECs)

(ii) The net useful heat generated by the CHP RTGU earns one AEC per net MWh of useful heat transferred to a useful load

(b) A CHP RTGU may qualify as a RPS Class I generator and as an APS CHP system per the regulations in CMR 225 16.00 that pertain to APS CHP systems and the related Guidelines. In this case:

(i) The net MWh of electricity generated by the CHP RTGU earns Class I RECs

(ii) The net MWh of electricity and the net MWh of useful heat generated by the unit earn AECs

This guideline addresses only the portion of 3(H)(23) applicable to the thermal output of the CHP RTGU which will earn APS RTGU AECs.

For guidance on metering and the calculation of RPS Class I RECs consult the RPS Class I Regulations 225 CMR 14.003.


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3 225 CMR 14.00 can be found at http://www.mass.gov/courts/docs/lawlib/220-229cmr/225cmr14.pdf

(24) Intermediate, Fired RTGUs, which Generate a Hot Heat Transfer Fluid and/or Steam

The major components of an Intermediate, Fired RTGUs include:

(a) Primary heat generator (e.g. boiler or engine or fuel cell)

(b) Combustion air and exhaust system

(c) Fuel storage and delivery system

(d) Burner system

(e) Pollution control systems

(f) Controls

(g) Data Acquisition System (DAS)

(h) Pumps, piping, fittings and heat exchangers which interconnect the RTGU to the useful thermal load

Figure 12. Metering Diagram for an Intermediate, Fired RTUG (Both Hot HTF and Steam)

Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for an Intermediate size fired RTGUs are shown.
(25) **AEC Formula for Intermediate, Fired RTGUs**

\[ \text{AECs} = (\text{RH} - \text{NUH} - \text{NRF}) \times M \]

*Notes: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh. The Higher Heating Value must be used for each fuel.*

Where:

\[ \text{RH} = \text{Renewable heat transferred to a useful load and is not directly metered} \]

Combustion modes for Fired RTGUs (all fuels must be eligible fuels)

- (a) Combustion of a single renewable fuel
- (b) Co-firing of more than one renewable fuel
- (c) Co-firing of one or more renewable fuels and one or more non-renewable fuel
- (d) Blending of one or more renewable fuels
- (e) Blending of one or more renewable fuels with one or more non-renewable fuels

*Note: Co-firing is switching between fuels without blending.*

(26) **RTGUs That Are Not CHP RTGUs**

For all combustion modes RH is obtained by means of the certified performance of RTGU which shows the renewable thermal output for the directly metered renewable thermal fuel input.

\[ \text{RH} = \text{RF} \times \text{RTGU EFF} \]

Where:

\[ \text{RF} = \text{Renewable Fuel} \]

\[ \text{RTGU EFF} = \text{The efficiency of the RTGU} \]

*Note: All certifications by the RTGU supplier must be submitted on the supplier’s letterhead to the department for review and approval.*

\[ \text{NUH} = \text{non useful thermal heat rejected to a heat sink} \]

*Note: For systems which includes a system that rejects heat directly generated to a heat sink (e.g. a radiator or cooling tower), the provider must either provide a method by which the heat rejected can be quantified by the independent verifier, or provide a method, such as a relay or switch, which is actuated by the operation of the heat rejection system and which transmits a signal to the data acquisition system to set to zero all AECs generated during any interval during which the heat rejection system has been running.*
NRF = The directly metered non-renewable fuel other than for combustion (e.g. grid electricity, or other) consumed by the RTGU

When the non-renewable fuel is grid electricity,

NRF = G / (0.44 * F)

Where:

G = Grid supplied electrical energy

F = the factor applied to the current 0.44 ISO-NE marginal grid efficiency to account for the improvements during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act

M = The current multiplier for GSHPs in the Department’s Guideline on AEC Multipliers for Renewable Thermal Generation Units

(27) Intermediate, Fired RTGUs that are CHP RTGUs

For all combustion modes:

RH = The CHP RTGU electricity generated is directly metered. RH is obtained by means of the certified performance map of the CHP RTGU showing the renewable thermal output for each 10 kW increment of over the full range of the RTGU’s generating capacity. The kW of the CHP RTGU must be recorded at intervals of no longer than 5 minutes.

NUH = Non useful thermal heat rejected to a heat sink

Note: For systems which includes a system that rejects heat directly generated to a heat sink (e.g. a radiator or cooling tower), the provider must either provide a method by which the heat rejected can be quantified or provide a method, such as a relay or switch which is actuated by the operation of the heat rejection system and will transmit a signal to the data acquisition system to set to zero all AECs generated during any interval during which the heat rejection system has been running.

NCNRF = The directly metered non-renewable fuel other than for combustion (e.g. grid electricity, or other) consumed by the RTGU

When the non-renewable fuel is grid electricity,

NRF = G / (0.44 * F)

Where:

G = Grid supplied electrical energy
F = The factor applied to the current 0.44 ISO-NE marginal grid efficiency to account for the improvements during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act

M = the current multiplier for GSHPs in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

(28) **Large, Fired RTGs which Generate a Hot Heat Transfer Fluid**

Major Components of a Large, Fired RTGU which generates hot heat transfer fluid include:

(a) Primary heat generator (e.g. boiler or engine or fuel cell)

(b) Combustion air and exhaust system

(c) Fuel storage and delivery system

(d) Burner system

(e) Pollution control systems

(f) Controls

(g) Data acquisition system (DAS)

(h) Pumps, piping, fittings and heat exchangers which interconnect the RTGU to the useful thermal load
(29) **Metering for Large, Fired RTGUs which Generate a Hot Heat Transfer Fluid**

**Figure 13. Metering Diagram for a Large Fired RTUG which Generates a Hot Heat Transfer Fluid**

![Diagram showing metering for large fired RTUGs](image)

*Note: Only the components directly related to either the required APS metering or otherwise related to the terms of the AECs formula for a Large Fired RTGU which Generates a Hot Heat Transfer Fluid are shown.*

(30) **AEC Formulae for a Large RTGU Generating a Hot Heat Transfer Fluid**

\[
\text{AECs} = (\text{RH} - \text{NUH} - \text{NCRF} - P_1 - P_e) \times M
\]

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

Where:

- RH = Renewable heat transferred to a useful load and is directly metered

Combustion modes for Fired RTGUs (all fuels must be eligible fuels):  
(a) Combustion of a single renewable fuel  
(b) Co-firing of more than one renewable fuel  
(c) Co-firing of one or more renewable fuels and one or more non-renewable fuel  
(d) Blending of one or more renewable fuels  
(e) Blending of one or more renewable fuels with one or more non-renewable fuels
General Formulae:

For RTGUs

$$AECs = (RH – NUH – NCNRF – P_e) * M$$

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

For CHP RTGUs

$$AECs = (RH – NUH – NCNRF – P_t – P_e) * M$$

*Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh*

Where:

**RH** = Is the renewable thermal energy transferred to a useful load

**RH for any interval** = $RH_{supply} – RH_{return}$

$RH_{supply} = \text{is the renewable thermal energy supplied from the RTGU to the useful thermal load(s)}$

$RH_{return} = \text{is the renewable thermal energy returned from the useful thermal load the RTGU}$

**NUH** = Non useful thermal heat rejected to a heat sink

*Note: For systems which includes a system that rejects heat directly generated to a heat sink (e.g. a radiator or cooling tower), the provider must either provide a method by which the heat rejected can be quantified or provide a method, such as a relay or switch which is actuated by the operation of the heat rejection system and will transmit a signal to the data acquisition system to set to zero all AECs generated during any interval during which the heat rejection system has been running.*

**NCNRF** = Any consumption of non-renewable fuel other than for combustion

When the NCNRF renewable fuel is grid electricity

**NCRF** = $G / (0.44 * F)$

*Where:

$G = \text{Grid supplied electrical energy}$

$F = \text{the factor applied to the current 0.44 ISO-NE marginal grid efficiency to account for the improvements during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act}$

$P_e = \text{Parasitic Electrical Energy (this applies only to a CHP RTGU, see Sections A(7) and A(8) for further details)}$
M = the current multiplier for GSHPs in the Department’s *Guideline on AEC Multipliers for Renewable Thermal Generation Units*

RH for Modes a, b, c, d:

\[ RH = \text{Is the renewable thermal energy transferred to a useful load} \]

\[ RH \text{ for any interval} = RH_{\text{supply}} - RH_{\text{return}} \]

\[ = HTFF \times Cp \times HTF \times (HTFST - HTFRT)/3.412E6 \text{ (Btu/MWh)} \]

RH for Mode e

RH for any interval:

\[ (\text{AvgCRF} / (\text{AvgCRF} + \text{AvgNRCF}) \times HTFF \times Cp \times HTF \times (HTFST - HTFRT)) / 3.412E6 \text{ (Btu/MWh)} \]

Where:

HTFF = Heat transfer fluid flow (lbs)

RF = The metered renewable fuel

NRF = Non-renewable fuel

Cp HTF = The constant pressure specific heat of the HTF (Btu/lb-°F)

HTST= HTF supply temperature (°F)

HTRT= HTF return temperature (°F)

CRF = Consumption of all renewable fuel (Btu)

CNRF = Consumption of all non-renewable fuel for combustion (Btu)

AvgRCF = Average consumption of all renewable fuel (Btu)

AvgNRCF = Average consumption of all non-renewable fuel for combustion (Btu)

(31) Large, Fired RTGU which Generate Steam

The major components of a Large, RTGU that generates steam are:

(a) A primary heat generator (e.g. an internal combustion engine, combustion turbine, or boiler)

(b) A fired heat recovery steam generator (HRSG)
(c) A Deaerator / Feedwater Heater(s)

(d) Pumps, piping, fittings and heat exchangers which connect the primary heat generator with the facility thermal distribution system and allow the transfer of thermal energy from the generation system to the facility thermal loads.

(e) Condensate return system

(f) Feedwater system

(g) Make up water system

(h) A blowdown heat recovery system

(i) Pollution controls

(j) Auxiliary Systems (e.g. heat rejection system (e.g. radiators or cooling towers), feedwater pumps, fans, blowers, fuel gas compressors)

(k) System controls

(l) Data acquisition and plant historian systems

(32) General Metering Requirements for Large RTGUs Generating Steam

The following guidelines are based on a generic basic configuration as shown in Figure 13. The Department will evaluate metering plans and submittals based on alternative configurations on a case-by-case basis

(a) Btu and kWh meters as shown Figure 13.

(b) Fuel metering is required when a renewable fuel will be blended with an non-renewable fuel

(c) Data acquisition systems (DAS) for recording, storing and transmitting of time stamped kWh, Btu, and fuel meter data.

Note: Any RTGU using a wood biomass fuel must comply with the added requirements as set forth in the APS Guideline on Biomass, Biogas, and Biofuels for APS Renewable Thermal Generation Units.
Note: The ideal steam and condensate return distribution system would require no makeup water, related feed water heating and minimal deaerating. However, this ideal typically cannot be achieved. The Department considers that for a well-designed and maintained closed steam distribution system, the amount of makeup water should not be more than 10% of the amount of steam generated. For this reason, for a closed system, the steam heating of the feedwater is considered to be parasitic and which is accounted for by basing the useful heat transferred on the difference between the enthalpy of the steam generated minus the enthalpy of the feedwater supplied.

(33) AEC Formula for Large, RTGUs Generating Steam

$$AECs = (RH - NUH - NRF - P_t - P_o) \times M$$

Where:

RH = Renewable heat transferred to a useful load
Combustion modes for Fired RTGUs (all fuels must be eligible fuels):

(a) Combustion of a single renewable fuel
(b) Co-firing of more than one renewable fuel
(c) Co-firing of one or more renewable fuels and one or more non-renewable fuel
(d) Blending of one or more renewable fuels
(e) Blending of one or more renewable fuels with one or more non-renewable fuels

General Formulae:

For RTGUs

\[ \text{AECs} = (\text{RH} - \text{NUH} - \text{NRF} - P_t) \times M \]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh

For CHP RTGUs

\[ \text{AECs} = (\text{RH} - \text{NUH} - \text{NRF} - P_t - P_e) \times M \]

Note: All terms are the cumulative as-metered values. Unless otherwise indicated, all units in MWh

Where:

\( \text{RH} \) = Is the renewable thermal energy transferred to a useful load

\( \text{RH for any interval} = \text{RH}_{\text{supply}} - \text{RH}_{\text{return}} \)

\( \text{RH}_{\text{supply}} \) = Is the renewable thermal energy supplied from the RTGU to the useful thermal load(s)

\( \text{RH}_{\text{return}} \) = Is the renewable thermal energy returned from the useful thermal load the RTGU

\( \text{NUH} \) = non useful thermal heat rejected to a heat sink

Notes:

1) For systems which includes a system that rejects heat directly generated to a heat sink (e.g. a radiator, cooling tower or blow down drain), the provider must either provide a method by which the heat rejected can be metered or otherwise quantified or provide a method, such as a relay or switch which is actuated by the operation of the heat rejection system and will transmit a signal to the data acquisition system to set to zero all AECs generated during any interval during which the heat rejection system has been running.

2) Thermal energy in a boiler blowdown stream that is recovered and transferred to a useful load (e.g. heating make up or feedwater) is considered to be thermal energy transferred to a useful load.

\( \text{NCRF} \) = Any consumption of non-renewable fuel other than for combustion
When the NCNRF renewable fuel is grid electricity

\[ \text{NCRF} = \frac{G}{(0.44 \times F)} \]

Where:

\( G \) = Grid supplied electrical energy

\( F \) = the factor applied to the current 0.44 ISO-NE marginal grid efficiency to account for the improvements during life of a heat pump unit as projected in the Massachusetts Global Warming Solutions Act

\( P_e \) = Parasitic electrical energy (this applies only to a CHP RTGU, see Sections A(7) and A(8) for further details)

\( P_t \) = Parasitic thermal energy

*Note: Parasitic thermal energy consumption in RTGU steam generators is most often associated with motive steam to drive auxiliary systems e.g. feedwater pumps or compressors, absorption chillers used to pre-cool gas turbine inlet air, and feedwater heating.*

All efforts should be made to locate system Btu meters such that the consumption of thermal parasitic thermal energy is netted out. In the event that this cannot be accomplished the parasitic thermal energy of any auxiliary component with a demand above a To Be Determined value during nominal operating conditions will require a separate Btu meter and the energy required to operate them is not metered or included in the determination of the AECs.

The ideal steam and condensate return distribution system would require no makeup water, related feedwater heating and minimal deaerating. However, this ideal typically cannot be achieved. The Department considers that for a well-designed and maintained closed steam distribution system, the amount of makeup water should not be more than 10% of the amount of steam generated. For this reason, for a closed system, the steam heating of the feedwater is considered to be parasitic and which is accounted for by basing the useful heat transferred on the difference between the enthalpy of the steam generated minus the enthalpy of the feedwater supplied. Refer to Figure 13 for the placement of the Btu meter on the feedwater return after the deaerator.

Many steam generating and distribution systems are not closed in that there are locations where, by design, steam is released and are not returned as condensate. Examples of this are sparging (the direct injection of live steam into a process for the purpose of heating) or the use of steam as a reactant in a process.

Whenever the pounds of makeup water are greater than than 10% of the pounds of steam generated due to the reasons listed above or others as are approved by the Department on a case-by-case basis, the temperature to be used for the feedwater supply will be the temperature of the combined flows of the condensate return and the makeup water.
M = the current multiplier for GSHPs in the Department’s Guideline on AEC Multipliers for Renewable Thermal Generation Units

RH for Modes a, b, c, d:

RH = Is the renewable thermal energy transferred to a useful load

RH for any interval = RH\_supply – RH\_return

= (SF\*hS – FW*hFW) / 3.412E6 (Btu/MWh)

RH for Mode e:

RH for any interval is:

\[(\text{AvgCRF} / (\text{AvgCRF} + \text{AvgNRCF}) \times (\text{SF}\*\text{hS} – \text{FW}\*\text{hFW}) / 3.412E6 \text{ (Btu/MWh)}\]

Where:

SF = Steam generated (lbs.)

FW = Total feedwater flow (lbs.)

hS = Feedwater enthalpy (Btu/lb)

hFW = Feed water enthalpy (Btu/lb)

RF = The metered renewable fuel

NRF = Non-renewable fuel

Cp HTF = The constant pressure specific heat of the HTF (Btu/lb-°F)

HTST = HTF supply temperature (°F)

HTRT = HTF return temperature (°F)

CRF = Consumption of all renewable fuel (Btu)

CNRF = Consumption of all non-renewable fuel for combustion (Btu)

AvgRCF = Average consumption of all renewable fuel (Btu)

AvgNRCF = Average consumption of all non-renewable fuel for combustion (Btu)
4. **Miscellaneous**

The Department may permit an exception from any provision of this Guideline for good cause.